# **Tutorial for Harmonic Analysis using CAEPIPE**

#### The following are the Steps to perform Harmonic Analysis using CAEPIPE.

#### General

 A harmonic analysis is performed to determine the response of a piping system to sinusoidal loads. Harmonic forces can arise from unbalanced rotating equipment, acoustic vibrations caused by reciprocating equipment, flow impedance, and other sources. These forces can be damaging to a piping system if their frequency is close to the piping system's natural frequency, thereby introducing resonant conditions.

It is feasible that multiple harmonic loads may be applied simultaneously at different locations of a piping system. More complex forms of vibration, such as those caused by the fluid flow, may be considered as superposition of several simple harmonics, each with its own frequency, magnitude, and phase.

• A harmonic analysis uses the results from the modal analysis to obtain a solution. A single damping factor is used for all modes.

First, the maximum response for each harmonic load is obtained separately. Then, the total response for multiple simultaneous harmonic loads is determined by combining the individual responses. The combination method may be specified as the Root Mean Square (RMS) or Absolute Sum. Even in the case of a system with a single harmonic load, the said combination is always carried out, so that the resulting solution becomes an "unsigned" case. For an unsigned case, the actual values for displacements, element forces and moments, etc. computed internally by CAEPIPE prior to such combination can be +ve or -ve for the dynamic event.

#### Step 1:

Attached is a CAEPIPE model for Harmonic Analysis. For this model, let us assume the following.

- 1. Node 5 is connected to a Tank.
- 2. Node 40 is connected to a Pump Suction Nozzle.



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#	Name	Descript	tion		Ty pe	Density (lb/in3)	Nu	Joint facto	: Yield r (psi)	Tensi (psi)	le #	Temp (F)	E (psi)	Alpha (in/in/F	Allowat ) (psi)	ole
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2											2	-200	29.7E+	6 8.47E-I	6 20000	_
											3	-100	29.0E+	6 8.75E-I	6 20000	
											4	70	28.3E+	6 9.11E-I	6 20000	
											5	200	27.6E+	6 9.34E-I	6 20000	
											6	300	27.0E+	6 9.47E-I	5 20000	- 1
											7	400	26.5E+	6 9.59E-I	6 19300	_
											8	500	25.8E+	6 9.70E-I	6 17900	_
											9	600	25.3E+	6 9.82E-I	6 17000	
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1	6	6"	STD	6.6248	0.28				11		2.559	1	-		-	
2	8	8"	STD	8.6248	0.322	2										
3	10	10"	STD	10.75	0.36	5										
	-	1										- 1				_
1-0-	Caepipe	e : Load	s (2) -	[harmoni	canaly	/sis.mo	d (c: <sup>\</sup>	\tutoria	ls\harm	nonica	nalysis	;)]		—		$\times$
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			<b>E</b>	) A		1			->							
#	Name	T1 (F)	P1 (psi)	Desg.T (F)	Des (psi)	g.Pr.	Spe grav	ecific vity	Add.W (lb/ft)	/gt. V L	Vind .oad 1	Wir Loa	nd V ad 2 L	Vind .oad 3	Wind Load 4	
1	_1	365	145	365	145		1.0									
2	L2	500	464	500	464		1.0									

## Step 2:

The harmonic load can be imposed as a Force (FX/FY/FZ) at a specified frequency and phase angle. You may be able to get more information on the harmonic loading (mass, rpm, etc.) from the manufacturer of the equipment.

For this Tutorial, the following assumptions are made.

- 1. Frequency of the rotating equipment = 14.5 Hz.
- 2. Force in Global Z Direction = FZ = 9000 lb.

The above parameters are entered for analysis by creating a "Data" type called "Harmonic Load" through Layout window > Misc > Data types... at Node 40. See snap shot below for details.

Harmonic load at node 40	×
Frequency 14.5	(Hz)
Phase	(deg)
FX (lb) FY (lb)	FZ (lb) 9000
OK Cancel	

#### Step 3:

Define "Percentage of Damping" and "Combination" method for Harmonic analysis through CAEPIPE Layout window > Loads > Harmonic...

Harmonic Analysis $ imes$
Damping 🚺 (%)
Combination C Root Mean Square C Absolute Sum
OK Cancel

#### Step 4:

Then, include "Harmonic response" for analysis through Layout window > Loads > Load cases.

## Step 5:

Save the model and perform the analysis through Layout window > File > Analyze. CAEPIPE will apply these loads to compute the response of the piping system by performing a Harmonic analysis along with other load cases defined in the piping system.

#### Step 6:

Upon successful analysis, CAEPIPE will now show a "Load case" with name "Harmonic response" under "Support Loads", "Displacements", "Element forces" and "Support load summary" results.

Load cases (5)	×					
☑ Sustained (W+P)	🔲 Design (W+PD+TD)					
Empty Weight (W)	🔽 Modal analysis					
💌 Expansion (T1)	🔽 Harmonic response					
🔽 Operating (W+P1+T1)						
OK Cancel	All None					

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#	Nod	e Ta	g F	X(lb)	FY (lb)	FZ (I	b)	MX (ft-lb)	) MY (f	t-lb)	MZ (ft-	lb)					
1	5		39	9600	16	217		62	2591	43	2870						
2	40		53	33	1493	2	Í	166	3418	3	7354						
3	125		32	2	64	48		315	235		143						
File	Image: Displacements: Harmonic response - [harmonicanalysis.res (c:\tutorials\harmonicanalysis)] — — X   File Results View Options Window Help																
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#					Displace	ments (g	(lobal)										
	Node	e X (ind	ch)	Y (inch)	Z (inch)	XX (0	deg) \	rY (deg)	ZZ (deg	<u>)</u>							
1	5	0.000	)	0.000	0.000	0.000		0.0002	0.0000	-							
2	10	2.844		0.000	0.000	0.000	)7   (	J.4690	0.0812	-11							
3	15A	0.333	} .	0.005	0.000	0.008	34   1	1.9630	0.1675	-11							
4	158	0.845	)	0.010	0.007	0.034	48   1 - 0   4	1.7134	0.0566	- 4							
5	20A	0.053	; ,	0.009	0.071	0.03	08    40	J.7013 2.2204	0.2405	- 1							
7	208	0.000	,	0.033	0.073	0.074	46   L 72   L	J.3394 19179	0.1035	- 1							
6	95	0.000	,	0.120	0.237	0.007	$\frac{1}{22}$	J.2172	0.0301	- 1							
a	50	0.000	, ,	0.115	0.150	0.007	72 1	1.2172	0.0301	- 1							
10	30		,	0.103	0.135	0.007	71 (	1 2 3 7 1	0.0301	- 1							
11	35		, 1	0.102	0.025	0.00		1 3 3 5 9	0.0413	- 1							
12	38	0.000	, 1	0.001	0.404	0.000	14 (	1.0685	0.0140	- 1							
13	40	0.000	)	0.000	0.407	0.000		0.0000	0.0000	- 1							
		0.000		0.110	0.100	0.04		0.01.70	0.0004								
1-0-1	Caepipe	: Pipe for	ces in lo	ocal coordina	tes: Harmon	ic response	e - [harm	onicanalysis.	res (c:\tutor	ials\har	monicanal	ysis)]			- 1	o x	
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#	Node /	Axial (Ib)	y She (lh)	ar z Sheai	<sup>r</sup> Torsio Moment	n(ft-lb) ISIE	Inpla	ane(ft-lb) at SIE	Outplaı Moment	ne(ft-lb) LSIE	) Flex	k Fac FEo	tors FFt	SL+SO	(nsi)	SL+SO	I
1	5 10	217	(16) 16 16	39600	2870	01	62 128		259143	0				105765	25995	4.07	
2	10 15A	248 248	82 82	17395	2870		128		216060					90281 4114	25995	3.47 0.16	
3	15A	283	176	13490	2870	1.00	920	2.54	5728	2.12	7.95	7.95		5896	25995	0.23	
4	15B	307	911	299	22590	1.00	13992	2.34	787	2.12	1.30	7.30		12175	25995	0.57	
5	20A 20A	431 1650	1659	1567	22590	1.00	3320	2.54	4887	2.12	7.95	7.95		11643	25995	0.42	
6	20B	1627	110	226	2928	1.00	5932 5932	2.54	24549	2.12	7.95	7.95		11821	25995	0.71	
	45	1627	110	226	2928		7239		27228					13165	25995	0.51	
7	45   25	1599 1599	1042 1042	3284 3284	2928 2928	1.39	7239 -24286	2.00	27228 6306	1.00				13162 16800	25995 25995	0.51	
8	25 50	517 517	883 883	8919 8919	166 166	1.39	-40390	2.00	6176 32400	1.00				26199 14779	25995 25995	1.01	

💷 Caepipe : Support loa	ad summary	for anchor a	at node 5 -	[harmonica	nalysis.res (o	: –		×				
File Results View Op	otions Win	idow Help	)									
Load combination	FX (lb)	FY (lb)	FY (lb) FZ (lb) MX (ft-lb)		MY (ft-lb)	MZ (ft-lb)						
Sustained	-24	233	-98	-1783	-871	-185						
Operating1	195	799	-106	-3998	4147	-2077						
Sustained+Harmonic	39576	249	120	-1721	258271	2685						
Sustained-Harmonic	-39624	218	-315	-1845	-260014	-3056						
Operating1+Harmonic	39795	814	111	-3936	263289	793						
Operating1-Harmonic	-39406	783	-324	-4060	-254996	-4948						
Maximum	39795	814	120	-1721	263289	2685						
Minimum	-39624	218	-324	-4060	-260014	-4948						
Allowables	0	0	0	0 0		0						

# Step 7:

From the review of frequency results of CAEPIPE, it is noted that one of the natural frequencies of this piping system (i.e., frequency for Mode 10 shown in yellow highlight in the snap shot below) is close to the rotating equipment frequency of 14.5 Hz.

-0-1	Caepipe : Fre	quencies - [	[harmonic	analysis.re	s (c:\tutor	ials\harmo	nicanalysi	. –		$\times$
File	Results Vi	iew Optior	ns Wind	ow Help	)					
#	Frequency	Period	Partic	ipation fa	actors	Modal n	nass/Tot	tal mass		
	(Hz)	(second)	×	Y	Z	×	Y	Z		
1	1.473	0.6788	0.0528	0.5034	-3.3441	0.0001	0.0130	0.5743		
2	2.502	0.3998	-1.4706	-0.6359	-0.1343	0.1111	0.0208	0.0009		
3	3.141	0.3184	-2.4648	0.5853	0.0084	0.3120	0.0176	0.0000		
4	3.702	0.2702	0.2151	3.1984	0.3743	0.0024	0.5254	0.0072		
5	3.763	0.2657	0.1984	1.1702	0.1233	0.0020	0.0703	0.0008		
6	5.048	0.1981	-0.1425	0.3283	0.1984	0.0010	0.0055	0.0020		
7	5.539	0.1805	-0.0228	-0.0144	-0.7486	0.0000	0.0000	0.0288		
8	5.901	0.1695	0.1759	0.0126	0.1174	0.0016	0.0000	0.0007		
9	8.568	0.1167	-1.3319	0.0917	0.0886	0.0911	0.0004	0.0004		
10	14.553	0.0687	-1.0193	-0.0477	-0.0036	0.0534	0.0001	0.0000		
11	16.917	0.0591	0.1139	-0.9348	-0.0311	0.0007	0.0449	0.0000		
12	27.478	0.0364	-0.0830	-0.0110	-0.0420	0.0004	0.0000	0.0001		
13	51.942	0.0193	0.3711	-0.0917	-0.7651	0.0071	0.0004	0.0301		
14					Total	0.5828	0.6985	0.6454		

Due to closeness of Mode 10 frequency to the equipment frequency, it is observed that Mode 10 is excited on the piping system by the harmonic load, thereby creating a resonance. This can be seen graphically by plotting the mode shape corresponding to Mode 10 with frequency of "14.55 Hz" (figure shown on the left below) and the deflected shape for "harmonic response" case (figure shown on the right below). See snap shots for details.



#### Step 8:

In order to prevent piping failure due to resonance, it is important to suppress relevant modes by changing the stiffness of the piping system either by adding or by moving the existing piping supports. For example, for the layout shown above, a lateral restraint in X direction is added at Node 10 as the displacement in X direction is about 3" for Harmonic Response case prior to adding this X restraint. By adding this new support, the stiffness of the piping system is altered. This, in turn, removed the 10<sup>th</sup> frequency with "14.55 Hz", thereby ensuring that the natural frequency of the piping system is not close to the operating equipment frequency. See snap shots below.



	Caepipe : Fre	quencies -	harmonica	analysis_tr	x_10.res (o	:\tutorials\	harmoni	-		×	
File	Results Vi	ew Optior	ns Winde	ow Help	)						
4											
#	Frequency	Period	Partic	ipation fa	actors	Modal n	hass / Tot	tal mass			
	(Hz)	(second)	×	Y	Z	Х	Y	Z			
1	1.474	0.6783	-0.0075	-0.4970	3.3457	0.0000	0.0127	0.5749			
2	2.956	0.3382	-0.9955	0.9922	0.0814	0.0509	0.0506	0.0003			
3	3.581	0.2792	-1.5073	-1.3860	-0.1892	0.1167	0.0987	0.0018			
4	3.729	0.2682	0.0054	3.0516	0.3445	0.0000	0.4782	0.0061			
5	4.705	0.2125	-1.8744	0.4453	0.0945	0.1804	0.0102	0.0005			
6	5.101	0.1961	-0.4923	-0.2136	-0.1835	0.0124	0.0023	0.0017			
7	5.540	0.1805	0.0155	-0.0171	-0.7481	0.0000	0.0000	0.0287			
8	5.904	0.1694	-0.2497	-0.0065	-0.1154	0.0032	0.0000	0.0007			
9	8.585	0.1165	-1.3138	0.0878	0.0881	0.0886	0.0004	0.0004			
10	16.913	0.0591	-0.0654	0.9358	0.0313	0.0002	0.0450	0.0001			
11	27.472	0.0364	-0.1185	-0.0109	-0.0420	0.0007	0.0000	0.0001			
12	51.938	0.0193	0.3657	-0.0902	-0.7605	0.0069	0.0004	0.0297			
13					Total	0.4601	0.6985	0.6450			