## Tutorial on Pressure Design of Pipe and Pipe Fittings according to EN 13480-3 (2017)

Pressure Design of Pipe and Pipe Fittings can be performed using the modules built into CAEPIPE which are independent of the flexibility analysis.
These modules can be launched through Layout frame > Misc > Internal Pressure Design: EN 13480-3 and Layout frame > Misc > External Pressure Design: EN 13480-3 respectively.

## Note:

These modules perform Pressure Design of Pipe and Pipe Fittings ONLY using the equations given in the EN 13480-3 (2017) Code irrespective of the Analysis Code selected for flexibility analysis in CAEPIPE.

In case the flexibility analysis is performed with an Analysis Code other than EN 13480-3 (2017), the Pressure Design modules will use the material allowable stresses corresponding to the maximum temperature T1 through T10 entered in the CAEPIPE stress model.

## Tutorial on Internal Pressure Design of Pipe and Pipe Fittings

## Step 1:

Snap shots shown below present a sample stress model developed to show the Internal Pressure Design calculations performed by CAEPIPE.




| -llt Caepipe : Loads (1) - [InternalPressureDesign.mod (C:\Tutorials\02_Press... |  |  |  |  |  |  |  |  |  |  |  | - | $\square$ | $\times$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Eile Edit View Options Misc Window Help |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| - |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| \# | Name | $\begin{aligned} & T 1 \\ & (C) \\ & (C) \end{aligned}$ | $\begin{array}{\|l\|l\|l\|l\|l\|l\|l\|l\|} \hline \text { (bar) } \\ \hline \end{array}$ | $\begin{aligned} & \mathrm{T} 2 \\ & (\mathrm{C}) \end{aligned}$ | $\begin{aligned} & \hline \begin{array}{l} \text { P2 } \\ \text { (bar) } \end{array} \end{aligned}$ | $\begin{array}{\|l} \hline \text { Desg.T } \\ \text { (C) } \\ \hline \end{array}$ | $\begin{aligned} & \text { Desg. } \mathrm{Pr} . \\ & \text { (bar) } \end{aligned}$ | Specific gravity | $\begin{aligned} & \text { Add.Wgt. } \\ & (\mathrm{kg} / \mathrm{m}) \end{aligned}$ | Wind Load 1 | Wind Load 2 | Wind Load 3 | Wind Load 4 |  |
| 1 | 1 | 221 | 22.6 | 20 | -1.00 | 221 | 22.6 | 0.003 |  |  |  |  |  |  |
| 2 |  |  |  |  |  |  |  |  |  |  |  |  |  |  |


| 게= Caepipe : Bends (17) - [InternalPressureDesign.mod (C:\Tutorials\PressureDesign)] |  |  |  |  |  |  |  |  |  |  |  |  | - | $\square$ | $\times$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
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| -11 |  |  |  |  |  | $\square$ |  |  |  |  |  |  |  |  |  |
| \# | Bend Node | Radius (mm) | Rad. Type | $\left.\begin{array}{\|l\|l} \text { Thk } \\ (\mathrm{mm}) \end{array} \right\rvert\,$ | Bend Matl | Flex.F | $\begin{aligned} & \ln \mathrm{Pln} \\ & \mathrm{SIF} \end{aligned}$ | $\begin{array}{\|l\|} \hline \text { Out Pln } \\ \text { SIF } \end{array}$ | Int. Node | Angle (deg) | Int. Node | Angle <br> (deg) |  |  |  |
| 1 | 20 | B56 | Short |  |  |  |  |  |  |  |  |  |  |  |  |
| 2 | 40 | 533 | Long |  |  |  |  |  |  |  |  |  |  |  |  |
| 3 | 70 | 533 | Long |  |  |  |  |  |  |  |  |  |  |  |  |
| 4 | 80 | 533 | Long |  |  |  |  |  |  |  |  |  |  |  |  |
| 5 | 210 | 305 | Long |  |  |  |  |  |  |  |  |  |  |  |  |
| 6 | 1010 | 356 | Short |  |  |  |  |  |  |  |  |  |  |  |  |
| 7 | 1030 | 533 | Long |  |  |  |  |  |  |  |  |  |  |  |  |
| 8 | 1060 | 533 | Long |  |  |  |  |  |  |  |  |  |  |  |  |
| 9 | 1070 | 533 | Long |  |  |  |  |  |  |  |  | $\checkmark$ |  |  |  |
|  | 1700 |  | $\|1 \ldots-\|$ |  |  |  |  |  |  |  |  |  |  |  |  |



## Step 2:

Internal pressure design calculations of pipe and pipe fittings according to EN 13480-3 are independent of lengths of elements defined in the CAEPIPE stress model. Hence, these calculations can be performed directly from the existing stress model developed for flexibility analysis. Equations used for performing Internal Pressure Design as per EN 13480-3 (2017) are provided at the end of this tutorial for reference.

Once the layout of the stress model as shown in the above snap shots is completed, the internal pressure design is performed through Layout window > Misc > Internal Pressure Design: EN 13480-3.
When executed, CAEPIPE automatically performs the pressure design calculations for Pipes, Elbows, Miters, Bends and Reducers for the entire stress model and displays the results as shown below.

It is observed that the ratios Uf1 and Uf2 are all less than 1.0, confirming that the Internal Pressure Design requirements of EN 13480-3 (2017) code are met for this stress model.

| File Options Window Help |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| \# | From | To | Element <br> Type | Des. Temp <br> (C) | Des.Press (bar) | $\begin{array}{\|l\|} \hline \text { All. Stress } \\ (\mathrm{N} / \mathrm{mm} 2) \end{array}$ | $\begin{aligned} & \hline 0 \mathrm{D} 1 \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{OD} 2 \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \text { Cor.All } \\ & (\mathrm{mm}) \end{aligned}$ | Radius (mm) | Cone Angle (deg) | $\begin{aligned} & \mathrm{ea} 1 \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{ea} 2 \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{ep} 1 \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{ep} 2 \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \text { Uf1 } \\ & \text { (ep 1/ea1) } \end{aligned}$ |  | $\wedge$ |
| 1 | 10 | 20 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 356 |  | 6 | 6 | 4.4417 | 4.4417 | 0.74 | 0.74 |  |
| 2 |  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 356 |  | 6 | 6 | 4.4824 | 2.4943 | 0.75 | 0.42 |  |
| 3 | 20 | 30 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 4 | 30 | 40 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
| 5 |  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 6 | 40 | 50 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 7 | 50 | 60 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 8 | 60 | 70 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
| 9 |  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 10 | 70 | 80 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
| 11 |  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 12 | 80 | 90 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 13 | 90 | 100 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 14 | 100 | 110 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 15 | 120 | 130 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 16 | 140 | 150 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 17 | 150 | 160 | Reducer | 221 | 22.6 | 132.8 | 406.4 | 355.6 | 1 |  | 8 | 6.7 | 6 | 3.4289 | 3.0003 | 0.51 | 0.50 |  |
| 18 | 160 | 170 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |  |
| 19 | 170 | 180 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |  |
| 20 | 180 | 190 | Reducer | 221 | 22.6 | 132.8 | 406.4 | 219.1 | 1 |  | 46 | 6.7 | 4.5125 | 4.4882 | 2.9294 | 0.67 | 0.65 |  |
| 21 | 190 | 200 | Pipe | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 |  |  | 4.5125 | 4.5125 | 1.8486 | 1.8486 | 0.41 | 0.41 | $\checkmark$ |

See the attached model "InternalPressureDesign.mod" for more details.

## Step 3:

The results shown above can also be printed to the printer or to a file using the option File > Print.


| aepipe Pressure Design (Internal) |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  | Page 1 |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Internal Pressure Design: EN 13480-3 (2017) (74) |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| From | To | Element Type | Des.Temp (C) | Des.Press (bar) | All.Stress ( $\mathrm{N} / \mathrm{mm} 2$ ) | $\begin{aligned} & \text { OD1 } \\ & (\mathrm{mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{OD2} \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \text { Cor.All } \\ & (\mathrm{mm}) \end{aligned}$ | $\begin{aligned} & \text { Radius } \\ & (\mathrm{mm}) \end{aligned}$ | Cone Angle (deg) | $\begin{aligned} & \text { ea1 } \\ & (\mathrm{mm}) \end{aligned}$ | $\begin{aligned} & \text { ea2 } \\ & (\mathrm{mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{ep1} \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{ep} 2 \\ & (\mathrm{~mm}) \end{aligned}$ | Uf1 (ep1/ea1) | Uf2 <br> (ep2/ea2) |  |
| 10 | 20 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 356 |  | 6 | 6 | 4.4417 | 4.4417 | 0.74 | 0.74 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 356 |  | 6 | 6 | 4.4824 | 2.4943 | 0.75 | 0.42 |  |
| 20 | 30 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 30 | 40 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 40 | 50 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 50 | 60 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 60 | 70 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 70 | 80 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 80 | 90 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 90 | 100 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 100 | 110 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 120 | 130 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 140 | 150 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 150 | 160 | Reducer | 221 | 22.6 | 132.8 | 406.4 | 355.6 | 1 |  | 8 | 6.7 | 6 | 3.4289 | 3.0003 | 0.51 | 0.50 |  |
| 160 | 170 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |  |
| 170 | 180 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |  |
| 180 | 190 | Reducer | 221 | 22.6 | 132.8 | 406.4 | 219.1 | 1 |  | 46 | 6.7 | 4.5125 | 4.4882 | 2.9294 | 0.67 | 0.65 |  |
| 190 | 200 | Pipe | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 |  |  | 4.5125 | 4.5125 | 1.8486 | 1.8486 | 0.41 | 0.41 |  |
| 200 | 210 | Elbow | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 | 305 |  | 4.5125 | 4.5125 | 2.3525 | 2.3525 | 0.52 | 0.52 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 | 305 |  | 4.5125 | 4.5125 | 2.3634 | 1.6025 | 0.52 | 0.36 |  |
| 210 | 220 | Pipe | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 |  |  | 4.5125 | 4.5125 | 1.8486 | 1.8486 | 0.41 | 0.41 |  |
| 1000 | 1010 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 356 |  | 6 | 6 | 4.4417 | 4.4417 | 0.74 | 0.74 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 356 |  | 6 | 6 | 4.4824 | 2.4943 | 0.75 | 0.42 |  |
| 1010 | 1020 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |
| 1020 | 1030 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |  |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |  |
| 1030 | 1040 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |  |


| Pressure Design (Internal) |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Internal Pressure Design: EN 13480-3 (2017) (74) |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |  |
| From | To | Element <br> Type | Des.Temp <br> (C) | Des.Press (bar) | All.Stress ( $\mathrm{N} / \mathrm{mm} 2$ ) | $\begin{aligned} & \text { OD1 } \\ & (\mathrm{mm}) \end{aligned}$ | $\begin{aligned} & \mathrm{OD} 2 \\ & (\mathrm{~mm}) \end{aligned}$ | Cor.All (mm) | Radius (mm) | Cone Angle (deg) | ea1 <br> (mm) | $\begin{aligned} & \mathrm{ea2} \\ & (\mathrm{~mm}) \end{aligned}$ | $\begin{aligned} & \text { ep1 } \\ & (\mathrm{mm}) \end{aligned}$ | $\begin{aligned} & \text { ep2 } \\ & (\mathrm{mm}) \end{aligned}$ | Uf1 (ep1/ea1) | Uf2 <br> (ep2/ea2) |
| 1040 | 1050 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |
| 1050 | 1060 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |
| 1060 | 1070 | Elbow | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7316 | 3.7316 | 0.62 | 0.62 |
|  |  | Bend | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 | 533 |  | 6 | 6 | 3.7468 | 2.6225 | 0.62 | 0.44 |
| 1070 | 1080 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |
| 1080 | 1090 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |
| 1090 | 1100 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |
| 1110 | 1120 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |
| 1130 | 1140 | Pipe | 221 | 22.6 | 132.8 | 355.6 | 355.6 | 1 |  |  | 6 | 6 | 3.0003 | 3.0003 | 0.50 | 0.50 |
| 1140 | 1150 | Reducer | 221 | 22.6 | 132.8 | 406.4 | 355.6 | 1 |  | 8 | 6.7 | 6 | 3.4289 | 3.0003 | 0.51 | 0.50 |
| 1150 | 1160 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |
| 1160 | 1170 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |
| 1170 | 1180 | Reducer | 221 | 22.6 | 132.8 | 406.4 | 219.1 | 1 |  | 46 | 6.7 | 4.5125 | 4.4882 | 2.9294 | 0.67 | 0.65 |
| 1180 | 1190 | Pipe | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 |  |  | 4.5125 | 4.5125 | 1.8486 | 1.8486 | 0.41 | 0.41 |
| 1190 | 1200 | Elbow | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 | 305 |  | 4.5125 | 4.5125 | 2.3525 | 2.3525 | 0.52 | 0.52 |
|  |  | Bend | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 | 305 |  | 4.5125 | 4.5125 | 2.3634 | 1.6025 | 0.52 | 0.36 |
| 1200 | 1210 | Elbow | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 | 305 |  | 4.5125 | 4.5125 | 2.3525 | 2.3525 | 0.52 | 0.52 |
|  |  | Bend | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 | 305 |  | 4.5125 | 4.5125 | 2.3634 | 1.6025 | 0.52 | 0.36 |
| 1210 | 1220 | Pipe | 221 | 22.6 | 132.8 | 219.1 | 219.1 | 1 |  |  | 4.5125 | 4.5125 | 1.8486 | 1.8486 | 0.41 | 0.41 |
| 170 | 1600 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |
| 1600 | 1610 | Elbow | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 | 610 |  | 6.7 | 6.7 | 4.2629 | 4.2629 | 0.64 | 0.64 |
|  |  | Bend | 221 | 22.6 | 132.8 | 406.4 | 406.4 |  | 610 |  | 6.7 | 6.7 | 4.2803 | 2.9976 | 0.64 | 0.45 |
| 1610 | 1620 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 |  |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |
| 1160 | 1850 | Pipe | 221 | 22.6 | 132.8 | 406.4 | 406.4 | 1 |  |  | 6.7 | 6.7 | 3.4289 | 3.4289 | 0.51 | 0.51 |
| 1620 | 1800 | Reducer | 221 | 22.6 | 132.8 | 508 | 406.4 | 1 |  | 18 | 8.625 | 6.7 | 4.2861 | 3.7079 | 0.50 | 0.55 |
| 1800 | 1810 | Pipe | 221 | 22.6 | 132.8 | 508 | 508 | 1 |  |  | 8.625 | 8.625 | 4.2861 | 4.2861 | 0.50 | 0.50 |
| 1810 | 1820 | Pipe | 221 | 22.6 | 132.8 | 508 | 508 | 1 |  |  | 8.625 | 8.625 | 4.2861 | 4.2861 | 0.50 | 0.50 |
| 1820 | 1830 | Pipe | 221 | 22.6 | 132.8 | 508 | 508 | 1 |  |  | 8.625 | 8.625 | 4.2861 | 4.2861 | 0.50 | 0.50 |
| 1830 | 1840 | Pipe | 221 | 22.6 | 132.8 | 508 | 508 | 1 |  |  | 8.625 | 8.625 | 4.2861 | 4.2861 | 0.50 | 0.50 |

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InternalPressureDesign
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## Tutorial on External Pressure Design of Pipe and Pipe Fittings

External Pressure Design module will function ONLY when the stress layout is defined with negative pressure (such as vacuum pressure).
This module first calculates collapse pressure (same as buckling pressure), which is a function of span length "L" between the stiffeners placed on the piping (shown in figures below). Since the collapse (buckling) mode of deformation for a pipe element between two adjacent stiffeners is restrained by these stiffeners, shorter the span length $L$ between the stiffeners, higher the collapse (buckling) pressure.
The External Pressure Design module assumes that a stiffener is located at each node of the CAEPIPE model. Hence, ensure that nodes are defined in CAEPIPE model only at locations where the stiffeners are attached to the piping. Even nodes where flanges or certain types of supports that restrain the collapse (buckling) mode of deformation should be included as "stiffener locations". All other nodes at which the collapse (buckling) mode of deformation is not restrained (such as resting supports) should not be included in the CAEPIPE model for external pressure design calculations. In other words, the CAEPIPE stress model (that was developed for flexibility analysis) needs to be edited before performing the external pressure design.


Single Pipe


Pipe with bend


Pipe with flange connections


Pipe with bend or elbow with ' $L$ ' measured on extrados


Pipe with mitre with ' $L$ ' measured on extrados

## Step 1:

The procedure given below will help in retaining ONLY those nodes of the CAEPIPE stress model (originally developed for flexibility analysis) prior to External Pressure Design calculations.

- Create a copy of the existing CAEPIPE stress model (developed for flexibility analysis).
- At whichever node the collapse (buckling) mode of deformation is NOT restrained, navigate to that element node in the layout window and use the option "Combine..." through Layout window > Edit. This action will remove that node by combining the two adjacent elements.
- Repeat Step 2 above and remove all other nodes where there are NO stiffeners or flanges or supports [that restrain the collapse (buckling) mode] defined.
- Upon completion, save the model.

Snap shots shown below present a sample model developed to show the External Pressure Design calculations performed by CAEPIPE. As stated above, a copy of the original stress model was made and the model has been edited to include only those nodes on pipe where stiffeners, flanges and supports (that are equivalent to stiffeners from the point of view of restraining collapse mode of deformation) are attached.


Eile Edit View Options Misc Window Help


| \# | Name | T1 <br> (C) | P1 <br> (bar) | T2 <br> (C) | P2 <br> (bar) | Desg. T <br> (C) | Desg.Pr. <br> (bar) | Specific <br> gravity | Add.Wgt. <br> (kg/m) | Wind <br> Load 1 | Wind <br> Load 2 | Wind <br> Load 3 | Wind <br> Load 4 |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| 1 | L1 | 185 | 10.0 | 21.11 | -1.00 | 185 | 10.0 | 0.1 |  | $Y$ |  |  |  |
| 2 | L2 | 260 | 32.0 | 21.11 | -1.00 | 260 | 32.0 | 0.1 |  | $Y$ |  |  |  |
| 3 |  |  |  |  |  |  |  |  |  |  |  |  |  |



## Step 2:

Once the layout of the stress model as shown in the above snap shots is completed, the external pressure design is performed through Layout window > Misc > External Pressure Design: EN 13480-3.
When executed, CAEPIPE automatically performs the external pressure design calculations for Pipes, Miters, Elbows, Bends and Reducers for the entire stress model and displays the results as shown below.
It is observed that the ratio $\left[P_{d} /\left(\mathrm{KP}_{\mathrm{c}}\right)\right]$ is much higher than 1.0 throughout the stress model, confirming that the collapse (buckling) pressures $\mathrm{P}_{\mathrm{r}}$ calculated for all segments of the stress model are much higher than the corresponding peak negative pressures specified in the CAEPIPE model. In other words, the potential for any segment of this piping system to collapse (buckle) is very minimal.


## Step 3:

The results shown above can also be printed to the printer or to a file using the option File > Print.



## Design of pipe and pipe fittings under internal pressure according to EN 13480-3 (2017)

## Straight Pipes

The minimum required wall thickness for a straight pipe without allowances and tolerances, ep, is calculated from equation 6.1-1 and 6.1-3 depending on the ratio between inner and outer diameter as follows:
For $D_{0} / D_{i}<=1.7$

$$
e p=\frac{P_{c} D_{0}}{2 f z+P_{c}}
$$

For $D_{0} / D_{i}>1.7$

$$
e p=\frac{D_{o}}{2}\left[1-\sqrt{\frac{f z-p_{c}}{f z+p_{c}}}\right]
$$

where,
$D_{0}=$ outside diameter of pipe
$D_{i}=$ inside diameter of pipe $=D o-2 \times e_{n}$
$e_{\mathrm{n}}=$ nominal wall thickness of pipe
$f=$ Allowable stress for material at maximum temperature
$z=$ weld efficiency factor $=1.0$
$\mathrm{p}_{\mathrm{c}}=$ maximum internal pressure $=$ maximum of CAEPIPE input pressures P1 through P10
$e_{p}=$ minimum required wall thickness

## Elbows

The minimum required wall thickness of the intrados and the extrados of the elbow without allowances and tolerances, ep1 / ep2, is calculated from equation B.4.1-3

$$
\begin{gathered}
\mathrm{ep} 1=\mathrm{ep} 2=\mathrm{e} . \mathrm{B} \\
B=\frac{D_{0}}{2 e}-\frac{R}{e}+\sqrt{\left[\frac{D_{0}}{2 e}-\frac{R}{e}\right]^{2}+2 \frac{R}{e}-\frac{D_{0}}{2 e}}
\end{gathered}
$$

where
$D_{0}=$ outside diameter of elbow
$\mathrm{e}=$ minimum required wall thickness of corresponding straight pipe computed as per Eq. 6.1-1 or 6.1-3
$R=$ radius of the elbow
$e p 1=e p 2=$ minimum required wall thickness of the elbow

## Bends (formed by cold bending of straight pipes)

## Wall thickness of the intrados of the bend

The minimum required wall thickness of the intrados of the bend without allowances and tolerances, ep1, is calculated from equation B.4.1-1

$$
\mathrm{ep} 1=\mathrm{e} \cdot \mathrm{~B}_{\text {int }}
$$

$$
B_{i n t}=\frac{D_{0}}{2 e}+\frac{r}{e}-\left[\frac{D_{o}}{2 e}+\frac{r}{e}-1\right] \sqrt{\frac{\left(\frac{r}{e}\right)^{2}-\left(\frac{D_{o}}{2 e}\right)^{2}}{\left(\frac{r}{e}\right)^{2}-\frac{D_{o}}{2 e}\left(\frac{D_{o}}{2 e}-1\right)}}
$$

$\mathrm{r} / \mathrm{e}$ is calculated from

$$
\frac{r}{e}=\sqrt{\frac{1}{2}\left\{\left(\frac{D_{o}}{2 e}\right)^{2}+\left(\frac{R}{e}\right)^{2}\right\}+\sqrt{\frac{1}{4}\left(\left(\frac{D_{o}}{2 e}\right)^{2}+\left(\frac{R}{e}\right)^{2}\right)^{2}-\frac{D_{o}}{2 e}\left(\frac{D_{o}}{2 e}-1\right)\left(\frac{R}{e}\right)^{2}}}
$$

where
$D_{0}=$ outside diameter of bend
$D_{i}=$ inside diameter of bend $=D o-2 \times e_{n}$
$\mathrm{e}=$ minimum required wall thickness of corresponding straight pipe computed as per Eq. 6.1-1 or 6.1-3
$R=$ radius of the bend
$e_{p 1}=$ minimum required wall thickness of the intrados

## Wall thickness of the extrados of the bend

The minimum required wall thickness of the extrados of the bend without allowances and tolerances, ep2, is calculated from equation B.4.1-8

$$
\begin{gathered}
\mathrm{ep} 2=\mathrm{e} \cdot \mathrm{~B}_{\mathrm{ext}} \\
B_{\text {ext }}=\frac{D_{0}}{2 e}-\frac{r}{e}-\left[\frac{D_{o}}{2 e}-\frac{r}{e}-1\right] \sqrt{\frac{\left(\frac{r}{e}\right)^{2}-\left(\frac{D_{o}}{2 e}\right)^{2}}{\left(\frac{r}{e}\right)^{2}-\frac{D_{o}}{2 e}\left(\frac{D_{o}}{2 e}-1\right)}}
\end{gathered}
$$

r/e is calculated from

$$
\frac{r}{e}=\sqrt{\frac{1}{2}\left\{\left(\frac{D_{o}}{2 e}\right)^{2}+\left(\frac{R}{e}\right)^{2}\right\}+\sqrt{\frac{1}{4}\left(\left(\frac{D_{o}}{2 e}\right)^{2}+\left(\frac{R}{e}\right)^{2}\right)^{2}-\frac{D_{o}}{2 e}\left(\frac{D_{o}}{2 e}-1\right)\left(\frac{R}{e}\right)^{2}}}
$$

where
$D_{0}=$ outside diameter of bend
$D_{i}=$ inside diameter of bend $=D o-2 \times e_{n}$
$e=$ minimum required wall thickness of corresponding straight pipe computed as per Eq. 6.1-1 or 6.1-3
$R=$ radius of the bend
$e_{p 2}=$ minimum required wall thickness of the extrados

## Reducers



## Junction between the large end of a cone and a cylinder without a knuckle

The minimum required wall thickness ( $\mathrm{e}_{1}$ ) of the larger cylinder adjacent to the junction is calculated from Subsection 6.4.6.2 as the greater of $e_{\text {cyl }}$ and $e_{j}$ where $e_{j}$ is determined from

$$
\begin{align*}
& \beta=\frac{1}{3} \sqrt{\frac{D_{c}}{e_{j}}} \frac{\tan \alpha}{1+\frac{1}{\sqrt{\cos \alpha}}}-0.15  \tag{Eq.6.4.6-2}\\
& e_{j}=\frac{p_{c \beta D_{c}}}{2 f} \tag{Eq.6.4.6-1}
\end{align*}
$$

The value of $e_{j}$ is acceptable, if the value given by Eq. 6.4.6-1 is not less than that assumed in Eq. 6.4.6-2

$$
\begin{aligned}
& e_{c o n}=\frac{p_{c} D_{e}}{2 f Z+p_{c}} \frac{1}{\operatorname{COS}(\alpha)} \\
& \mathrm{e}_{\mathrm{cyl}}=\frac{\mathrm{p}_{\mathrm{c}} \mathrm{D}_{01}}{2 \mathrm{fZ}+\mathrm{p}_{\mathrm{c}}} \\
& e_{1}=\text { thickness of larger cylinder }=\max \left(e_{j}, e_{c y l}\right) \\
& e_{3}=\text { thickness of cone shell }=\max \left(e_{j}, e_{c o n}\right)
\end{aligned}
$$

where
$D_{\mathrm{e}}=$ outside diameter of the cone
$\mathrm{D}_{01}=$ outside diameter of the larger cylinder
$\mathrm{D}_{02}=$ outside diameter of the small cylinder
$D_{c}=$ mean diameter of the larger cylinder at the junction with the cone $=D_{01}-e_{n}$
$\mathrm{e}_{\mathrm{n}}=$ nominal wall thickness of the larger cylinder at the junction with the cone
$\alpha=$ cone angle
$\mathrm{e}_{1}=$ minimum required wall thickness for larger cylinder adjacent to the junction.
$e_{3}=$ minimum required wall thickness at cone.
$\mathrm{f}=$ Allowable stress for material at maximum temperature
$\mathrm{p}_{\mathrm{c}}=$ maximum internal pressure $=$ maximum of CAEPIPE input pressures P1 through P10
$Z=$ weld efficiency factor $=1.0$

## Junction between the small end of a cone and a cylinder without a knuckle

The minimum required wall thickness $\left(\mathrm{e}_{2}\right)$ of the small cylinder adjacent to the junction is calculated according to Subsection 6.4.8.2 as follows.

$$
s=\frac{e_{3}}{e_{j 2}}
$$

With $\mathrm{e}_{3}$ already determined in the earlier section, assume value of $\mathrm{e}_{\mathrm{j} 2}$ and calculate the values of $s, \tau$ and $\beta_{H}$ When s < 1.0 , then

$$
\tau=s \sqrt{\frac{s}{\cos \alpha}}+\sqrt{\frac{1+s^{2}}{2}}
$$

When $s>=1.0$, then

$$
\tau=1+\sqrt{s \frac{1+s^{2}}{2 \cos \alpha}}
$$

$\beta_{H}=0.4 \sqrt{\frac{D_{c}}{e_{\mathrm{j} 2}}} \frac{\tan \alpha}{\tau}+0.5$
$e_{j 2}=\frac{p_{c} D_{c} \beta_{H}}{2 f Z}$
The value of $\mathrm{e}_{\mathrm{j} 2}$ is acceptable, if the value given by Eq. 6.4.8-5 is not less than that assumed for Eq. 6.4.8-4

$$
\begin{gathered}
e_{c y l}=\frac{p_{c} D_{02}}{2 f Z+p_{c}} \\
e_{2}=\max \left(e_{j 2}, e_{c y l}\right)
\end{gathered}
$$

where
$\mathrm{D}_{02}=$ outside diameter of the small cylinder at the junction with the cone
$D_{c}=$ mean diameter of the small cylinder at the junction with the cone $=D_{02}-e_{n}$
$\mathrm{e}_{\mathrm{n}}=$ nominal wall thickness of the small cylinder at the junction with the cone
$\alpha=$ cone angle
$e_{2}=$ minimum required wall thickness of the small cylinder at the junction with the cone
$f=$ Allowable stress for material at maximum temperature
$\mathrm{p}_{\mathrm{c}}=$ maximum internal pressure $=$ maximum of CAEPIPE input pressures P1 through P10
$Z=$ weld efficiency factor $=1.0$

# Design of pipe and pipe fittings under external pressure according to EN 13480-3 (2017) 

## Pipes, Elbows, Mitre Bends and Reducers

## Interstiffener collapse

The thickness of the pipe within the unstiffened length $L$ shall not be less than that determined by the following.

$$
\begin{gathered}
P_{r} \geq k . P_{c} \\
P_{y}=\frac{S e_{a}}{R_{m}} \\
P_{m}=\frac{E_{t} e_{a} \varepsilon}{R_{m}} \\
\varepsilon=\frac{1}{n_{c y l^{2}}-1+\frac{Z^{2}}{2}}\left\{\frac{1}{\left(\frac{n_{c y l^{2}}}{Z^{2}}+1\right)^{2}}+\frac{e_{a^{2}}}{12 R_{m^{2}\left(1-v^{2}\right)}}\left(n_{c y l^{2}}-1+Z^{2}\right)^{2}\right\} \\
Z=\frac{\pi R_{m}}{L}
\end{gathered}
$$

using the calculated value of $\mathrm{Pm} / \mathrm{Py}, \mathrm{Pr} / \mathrm{Py}$ is determined from Table 9.3.2.1 of Subsection 9.3.2 where
$\mathrm{n}_{\mathrm{cyl}}=$ integer $>=2$ to minimize the value of $\mathrm{P}_{\mathrm{m}}$
$R_{m}=$ mean radius of the pipe
$\mathrm{L}=$ length between the stiffener, is calculated from CAEPIPE input as follows
for Pipe, $L$ = length of pipe (= distance between the "From" and "To" node of CAEPIPE)
for Elbow and Miter bend, $L=$ arc length measured on extrados of elbow and miter bend
for Reducer, $L=$ Length of the reducer
$E_{t}=$ Young's modulus of material at design temperature (max of CAEPIPE Temperature T 1 through T10)
$e_{a}=$ analysis thickness of reducer at smaller end $=e_{n}-$ corr.all - mill tolerance
$e_{n}=$ nominal thickness of reducer at smaller end
$\mathrm{k}=$ factor $=1.5$
$\mathrm{P}_{\mathrm{c}}=$ external pressure $=$ maximum negative CAEPIPE input pressures P 1 through P10
$S=$ elastic stress limits for pipe and stiffener
$=R_{\text {p0.2,t }}$ for non-austenitic steels
$=\left(R_{p 0.2, t} / 1.25\right)$ for austenitic steels
$R_{p 0.2, t}=$ minimum $0.2 \%$ proof strength at temperature of pipe
= 'f' for EN 13480 code and
= "Allowable stress" at temperature of pipe for other codes

## Additional check for Reducers

In addition to the above, as stated in Subsection 9.4.2 of EN 13480-3, the moment of inertia, $\mathrm{I}_{\mathrm{x}}$ taken parallel to the axis of the cylinder, of the part of the cone and cylinder with a distance of $\sqrt{D_{e q} \cdot e}$ on either side of the junction is not less than:

$$
I_{x}=0.18 D_{e q} L D_{s}^{2} \frac{p_{c}}{E_{t}} \leq I_{x a}
$$

where
$\mathrm{D}_{\text {eq }}=$ equivalent diameter $=\frac{\frac{D_{1}+D_{2}}{2}}{\cos (\alpha)}$
D1 = outside diameter of larger end of reducer
D2 = outside diameter of smaller end of reducer
$\alpha=$ cone angle of reducer input in CAEPIPE
$I_{\mathrm{xa}}=$ moment of inertia of reducer at smaller end
Ds = diameter of the centroid of the moment of inertia of the stiffening cross section calculated as shown below
$I_{\text {cone }}=\left(\sqrt{D_{\text {eq }} e_{1}} \cdot e_{1}\right)\left(\frac{D_{\text {mcon }}}{2}\right)^{2}=\left(A_{\text {cone }}\right)\left(\frac{D_{\text {mcon }}}{2}\right)^{2}$
$I_{C y l}=\left(\sqrt{D_{e q} e_{2}} \cdot e_{2}\right)\left(\frac{D_{m c y l}}{2}\right)^{2}=\left(A_{c y l}\right)\left(\frac{D_{m c y l}}{2}\right)^{2}$
$I_{\text {stiff }}=\left(A_{\text {cone }}+A_{\text {cyl }}\right)\left(\frac{D_{s}}{2}\right)^{2}$
From the above,
$I_{\text {cone }}+I_{C y l}=I_{\text {stiff }}$
and
$D_{s}=2 \sqrt{\frac{I_{\text {stiff }}}{\left(A_{\text {cone }}+A_{\text {cyl }}\right)}}$
e1 = analysis thickness of reducer at larger end $=e_{n 1}-$ corr.all - mill tolerance
$e 2=$ analysis thickness of reducer at smaller end $=e_{n 2}-$ corr.all - mill tolerance
$\mathrm{e}_{\mathrm{n} 1}=$ nominal thickness of reducer at larger end
$e_{n 2}=$ nominal thickness of reducer at smaller end

